



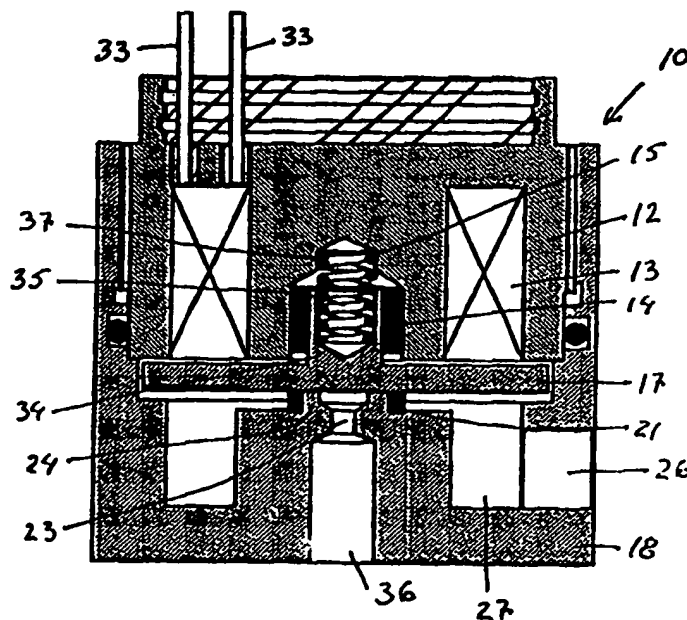
## INTERNATIONAL APPLICATION PUBLISHED UNDER THE PATENT COOPERATION TREATY (PCT)

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(54) Title: ELECTROMAGNETIC VALVE FOR GASEOUS FLUIDS

## (57) Abstract

An electromagnetic valve for a gaseous fluid comprising a combination of an electromagnetic circuit (13, 12, 17) which when activated generates an electromagnetic force for mechanical opening or closing of the valve, a movable disk formed member (17) at least part of which is included in said electromagnetic circuit and which disk formed member (17) is guided in a linear movement perpendicular to the surface of said disk formed member. The valve further comprises a valve seat (21, 24) co-operating with the disk formed member providing the mechanical closing and sealing of the valve, a resilient means (15) which acts on the disk formed member (17) and a metering means (23) in the form of a restriction arranged immediately downstream the valve seat creating sonic conditions for the gas flow which makes the gas flow with the valve open independent of fluctuations in the downstream pressure.



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## ELECTROMAGNETIC VALVE FOR GASEOUS FLUIDS

### TECHNICAL FIELD

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The present invention relates to an electromagnetic valve for gaseous fluids. This type of valve has a special application as injector valve for gaseous fuels for internal combustion engines. The invention therefore also relates to an internal combustion engine provided with one or several such valves.

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### BACKGROUND ART

15 The problems caused by atmospheric pollution by means of exhaust gases from internal combustion engines such as car motors incite scientists and car manufacturers to look for new fuels and alternate fuels like for instance compressed natural gas or liquefied petroleum gas (LPG) which potentially offer an important reduction of the emission of polluting gases and greenhouse gases.

20 The present solutions for injection and metering of these new fuels for an internal combustion engine offer however not yet performances equal to the solutions used for normal petrol.

25 The principle for the known fuel systems for metering and mixing a gaseous fuel with inlet air to an internal combustion engine are based on the Venturi effect or on continuous injection of fuel. These systems have, however, typically an important number of mechanical components and do not offer the response times adequate to satisfy the sometimes important variations in the demand for fuel by the engine.

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Thus, when these known injection systems for gaseous fuel are working together with for instance an associated exhaust gas purification system of the three-way catalytic converter type with lambda-probe control, their characteristics do not allow for the maintenance of the air to fuel ratio at the desired value during fast transitions in the engine load and speed in order to give an optimal purification of the exhaust gases.

#### DISCLOSURE OF THE INVENTION

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One object of the present invention is to provide an electromagnetic valve for general gaseous fluids which can be used in many industrial applications in which there is a need for a simple wear resistant valve with an extremely short response time.

15

Another object of the invention is to provide an electromagnetic injector valve for gaseous fuels to e.g. an internal combustion engine. Such an injector valve could be mounted on the inlet or inlet manifold of an internal combustion engine for injecting the fuel. Due to the short response time of the valve it could be operated in pulsating mode.

20

The total response time for a traditional injection system for gaseous fuel is due to the operation of the actuator or actuators which are used for metering the fuel, the time for the transport of the gas mixture through the inlet tube or manifold to the respective cylinder and the characteristics for the transport of the fuel through the feeding tubes to the manifold.

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The above and further objects are achieved by means of the present invention which is characterised in that the valve comprises:

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- an electromagnetic circuit which when activated generates an electromagnetic force for mechanical opening or closing of the valve,

5 - a movable disk formed member at least part of which is included in said electromagnetic circuit and which disk formed member is guided in a linear movement perpendicular to the surface of said disk formed member,

- a valve seat co-operating with the disk formed member providing the mechanical closing and sealing of the valve,

10

- a resilient means which acts on the disk formed member,

15 - a hole arranged downstream said valve seat the diameter of which hole preferably defines a sonic section for the gas flow which makes it possible to precisely adjust the quantity of gas let through as a function of the time during which the valve is in its open position irrespective of the downstream pressure.

20 Important advantages with a valve according to the invention is that it only has one (two if you count the spring) moving part(s), low inertia, extremely short response time and that it is very wear resistant.

25 When this type of valve is used as an injector valve for gaseous fuel for an internal combustion engine it makes possible the use of a pulsating mode for the injection of gas into the air flow introduced through the inlet or inlet manifold into the internal combustion engine allowing an optimal mixing of the two flows.

Due to the very simple but original design the reliability and life time are excellent.

30

## BRIEF DESCRIPTION OF THE DRAWINGS

Other objects, uses and advantages of this invention will be apparent from the reading of this description which proceeds with reference to the accompanying

5 drawings forming part thereof and wherein:

Figure 1 shows a section through the electromagnetic valve for a gaseous fluid according to the invention.

10 Figure 2 shows one embodiment of the lower surface of the disk formed member 17.

Figure 3 shows a section through the disk formed member 17 according to figure 2.

15

Figures 4 and 5 show two other embodiments of the disk formed member.

Figure 6 shows an internal combustion engine with an injector valve according to the invention.

20

## DETAILED DESCRIPTION OF THE INVENTION

25 The invention will now be further described by means of a particular embodiment.

Figure 1 shows a section through an electromagnetic valve for gaseous fluids according to the invention. This particular embodiment of the valve could be used as an electromagnetic injector valve for gaseous fuels for an internal combustion engine. Due to its characteristics it could in that particular application be controlled in order to give indirect, pulsating injection of gaseous fuels under

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pressure into the engine. Indirect injection refers to the injection mode according to which the gaseous fuel is added to the flow of air fed to the cylinder or cylinders. The valve could be mounted very close to the inlet of the motor in order to give a very short over all response time for the fuel system. On a multiple cylinder engine a common valve could be used for all cylinders with a distribution of the air-gas mixture through the manifold to each respective inlet. In another configuration one separate valve for each cylinder could be mounted on the manifold close to the respective inlet. This gives the advantage of individual metering the fuel to each respective cylinder.

10

The electromagnetic valve with the general designation 10 comprises a magnetic circuit having a fixed armature 12 and a disk formed member 17 forming a movable part of said magnetic circuit. It should be noted that only part of the disk formed member has to be able to conduct the magnetic field. A solenoid coil 13 is arranged in an annular groove in the fixed armature 12. Electrical connections 33 on the solenoid coil are arranged to receive electrical energy which allows the activation of the electromagnetical valve by generating a magnetic force which will displace the disk formed member 17.

20 An annular valve seat 24 is arranged to co-operate with the disk formed member and a resilient member in e.g. the form of a spring 15 is in this embodiment biasing the disk formed member into contact with the valve seat 24 when the valve is not activated. The valve is thus normally closed. A general purpose valve of this type could of course be designed to be normally open.

25

In the illustrated embodiment the disk formed member 17 is circular and is provided with a concentric cylindrical guiding member or shaft 34 fixed to and arranged perpendicularly to the upper surface of the disk formed member for guiding the movement of said disk formed member in a direction perpendicular to the surface of the disk.

30

The guiding member 34 of the disk formed member 17 is in this embodiment arranged in an appropriate slide bearing 14 at the center of the fixed armature 12. The bearing could be of the self lubricating type.

5 The resilient member 15 is in this case arranged between the guiding member and the fixed armature. In the embodiment illustrated in figure 1 the guiding member 34 forms a tube the cylindrical interior 35 of which is arranged to receive the lower part of said resilient member in the form of a spring 15. The upper part of the spring 15 is arranged in a cylindrical cavity 37 in the center of the  
10 armature 12. The cylindrical external surface of the guiding member is sliding in the bearing 14 and is guiding the disk formed member 17 in its movement.

In other embodiments of the valve according to the invention the guiding of the disk formed member could be achieved at the periphery of said disk. In that case  
15 no central guiding member 34, and co-operating slide bearing 14 will be necessary. The resilient means 15 could of course also be arranged at the periphery of the disk formed member and does not have to have the form of a spring. It could for instance be realized in the form of a ring of elastic material.

20 In order to keep the response time of the valve small it is important that there is always the same pressure on both sides of the disk thus also at start up when pressurised gas suddenly is entering the inlet 26 of the valve. This can be achieved by means of e.g. a by-pass channel in the body of the valve letting both sides of the disk communicate pneumatically. Appropriate holes in the disk  
25 could give the same result. The gas could also be allowed to circulate around the edge of the disk between the disk and the inner surface of the valve housing. In order to quickly equilibrate the pressure on the two sides of the disk formed member a thin disk provided with at least one radial slot starting from the periphery of the disk could be arranged on top of the disk formed member 17.

30

In a particular embodiment of the valve according to the invention as illustrated



in figures 1-3 for use as an injector valve the disk formed member is only moving 0.3 mm which means that the gas turbulence around the edge is of negligible importance.

- 5 The disk 17 could also have a considerably smaller diameter than is shown in figure 1 and be provided with impeller blades at the periphery.

In the illustrated embodiment the activation of the electromagnetic circuit will move the disk formed member upwards in figure 1 and thus open the valve.

10

An inlet opening 26 for gaseous fluid of fuel is arranged on the side of the body 18 of the injector valve. The inlet opening could of course be arranged differently.

- 15 The annular valve seat 24 is sealing against the lower surface of the disk formed member 17 at rest. Preferably a ring 21 of e.g. a somewhat elastic material like a polymer forms the seat. This allows on one hand the absorption of the energy of the shock when the disk is landing on the seat due to the deactivation of the solenoid 13, and on the other hand it allows for excellent sealing of the valve so  
20 that the gaseous fluid can not circulate from the inlet 26 through the valve to the outlet conduit 36 when the valve is closed.

- When the solenoid 13 is electrically activated, the disk formed member 17 at least part of which is included in the electromagnetic circuit moves from the  
25 lower position towards an upper position in figure 1 and allows the flow of gaseous fluid from the inlet conduit 26 via a coaxial annular cavity 27 arranged under the disk to further pass between the disk and the annular seat in the direction of the downstream passage 23 and the outlet conduit 36. The sectional area of the passage 23 of the valve is in this embodiment precisely defined in order to get a  
30 sonic flow of the gaseous fluid through the valve. This means that the flowrate will be independent of the pressure on the downstream side of the valve which is

very important in an application as injector valve as this pressure changes considerably during the engine working cycle. At the same time the opening between the seat 24 and the disk in the fully open position of the valve is defined so that it is much bigger than the area of the hole 23 which will allow for the sonic flow in the hole 23. Thus the flow of gaseous fluid only depends on the pressure of the gaseous fluid on the inlet to the valve 10.

In a preferred embodiment the disk formed member 17 is provided with grooves 25 arranged radially on its lower surface as shown in figure 2. These grooves have an inclined section which is causing a rotation of the disk 17 when the gaseous fluid circulate from the inlet conduit 26 through the valve towards the outlet conduit 36. The stepwise rotation of the disk 17 allows for a uniform wear of the seat 24, the disk 17 in the area where it touches the seat 24 in the closed position, the guiding member 34 on the disk 17 and the slide bearing 14.

The rotation of the disk could also be achieved by means of e.g. through holes having side walls inclined in relation to the lower and upper surfaces of the disk 17 or slots 28 arranged radially in the disk 17 also having an inclined section.

Outside the central area of the disk which is cooperating with the valve seat disk material could be removed in order to lower the mass of the disk 17.

When the valve is used as an injector valve for an internal combustion engine 38 the electrical conductors 33 of the solenoid 13 could be connected to a control unit 39 which delivers the control signal for activating the valve. Preferably a number of probes 40, 41, ...arranged on the engine are measuring relevant control parameters which are fed to the control unit in which a calculation according to some appropriate control algorithm is carried out and a resulting control signal is generated. This arrangement allows for the injection of a gaseous fuel with a variable duration and variable frequency, thus with a variable volume, and with a variable phase. The injector valve could consequently deliver fuel at

any particular moment of the cycle of the engine, even when the respective inlet valve(s) of the engine is (are) closed. The appropriate air-gas mixture is in such a case momentarily stored in the manifold waiting for the next opening of the respective valve.

## CLAIMS

1. Electromagnetic valve for a gaseous fluid characterised in that it  
5 comprises a combination of an electromagnetic circuit (13, 12, 17) which when  
activated generates an electromagnetic force for mechanical opening or closing  
of the valve, a movable disk formed member (17) at least part of which is  
included in said electromagnetic circuit and which disk formed member (17) is  
10 guided in a linear movement perpendicular to the surface of said disk formed  
member, a valve seat (21, 24) co-operating with the disk formed member  
providing the mechanical closing and sealing of the valve, a resilient means (15)  
which acts on the disk formed member (17), a metering means (23) in the form  
of a restriction arranged immediately downstream the valve seat creating sonic  
15 conditions for the gas flow which makes the gas flow with the valve open  
independent of fluctuations in the downstream pressure.

2. Electromagnetic valve according to claim 1, characterised in that said  
movable disk formed member (17) has a circular form and is provided with a  
20 concentric cylindrical guiding member or shaft (34) fixed to and arranged  
perpendicularly to the upper surface of the disk formed member (17) arranged in  
an appropriate slide bearing (14) at the center of the fixed armature (12) for  
guiding the movement of said disk formed member in a direction perpendicular  
to the surface of the disk.

25

3. Electromagnetic valve according to claim 2, characterised in that said  
slide bearing (14) does not need lubrication.

30

4. Electromagnetic valve according to any of the claims 1 to 3, **characterised in that** said valve seat (24) has an annular form and is sealing against the lower surface of the disk formed member (17) at rest.

5

5. Electromagnetic valve according to any of the claims 1 to 4, **characterised in that** said valve seat (24) is provided with a ring (21) of an elastic material which allows on one hand the absorption of the energy of the shock when the disk is landing on the seat due to the deactivation of the solenoid (13) thereby limiting the rebounds, and on the other hand allows for sealing of the valve when closed.

6. Electromagnetic valve according to any of the claims 1 to 5, **characterised in that** said resilient means (15) has the form of a spring arranged at the center of the disk formed member (17).

7. Electromagnetic valve according to any of the claims 1 to 6, **characterised in that** said disk formed member (17) is provided with grooves (25) arranged radially on its lower surface which grooves have an inclined section which is causing a rotation of the disk (17) when the gaseous fluid circulate from the inlet conduit (26) through the valve towards the outlet conduit (36).

25

8. Electromagnetic valve according to any of the claims 1 to 7, **characterised in that** said disk formed member (17) is provided with slots (28) arranged radially in the disk (17) having an inclined section which is causing a rotation of the disk (17) when the gaseous fluid circulate from the inlet conduit (26) through the valve towards the outlet conduit (36).

30

9. Electromagnetic valve according to any of the claims 1 to 8,  
characterised in that said disk formed member (17) is provided with at least  
5 one through hole (28) having side walls inclined in relation to the lower and  
upper surfaces of the disk (17) which will cause a rotation of the disk (17) when  
the gaseous fluid circulate from the inlet conduit (26) through the valve towards  
the outlet conduit (36).

10

10. Internal combustion engine provided with at least one injector valve for  
gaseous fuel characterised in that the injector valve comprises a combination  
of an electromagnetic circuit (13, 12, 17) which when activated generates an  
electromagnetic force for mechanical opening or closing of the valve, a movable  
15 disk formed member (17) at least part of which is included in said  
electromagnetic circuit and which disk formed member (17) is guided in a linear  
movement perpendicular to the surface of said disk formed member, a valve  
seat (21, 24) co-operating with the disk formed member providing the  
mechanical closing and sealing of the valve, a resilient means (15) which acts on  
20 the disk formed member (17), a metering means (23) in the form of a restriction  
arranged immediately downstream the valve seat creating sonic conditions for  
the gas flow which makes the gas flow with the valve open independent of  
fluctuations in the downstream pressure.

25

11. Internal combustion engine according to claim 10 characterised in that  
at least one probe (40, 41) is arranged on the engine to measure relevant  
control parameters and feed corresponding electrical signals to a control unit  
(39) arranged to calculate according to some appropriate control algorithm a  
resulting control signal which is fed to the electrical connections (33) on a  
30 activating coil (13) forming part of said electromagnetic circuit (13, 12, 17) which

allows the activation of the electromagnetic valve, making it possible to inject a predetermined amount of gaseous fuel at any particular moment of the cycle of the engine into the air fed to said engine.

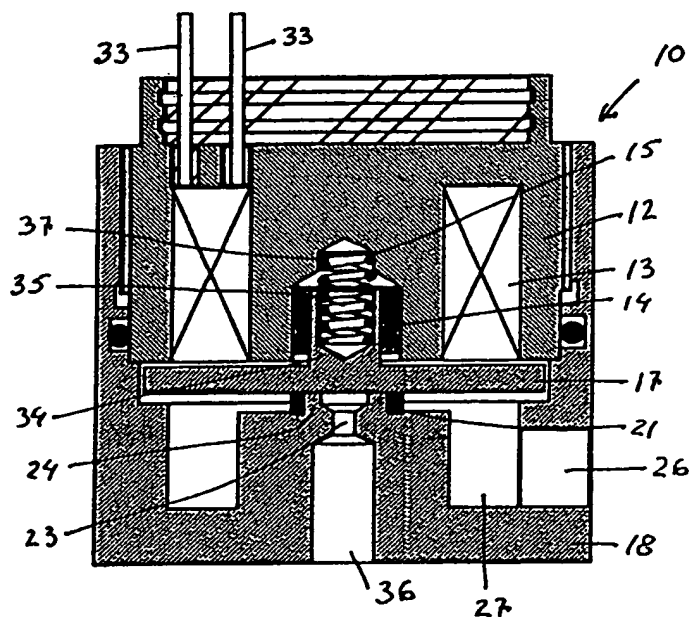


FIG 1

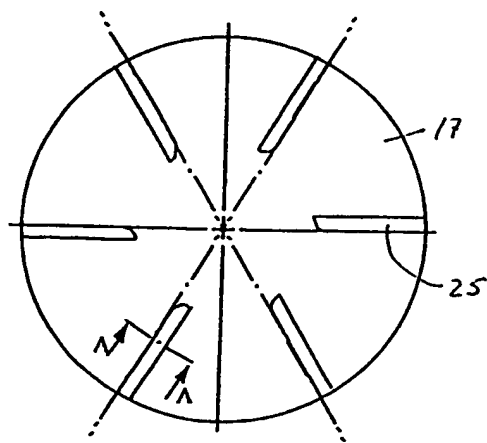


FIG 2

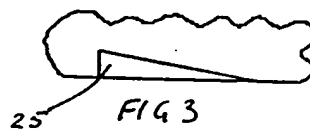


FIG 3

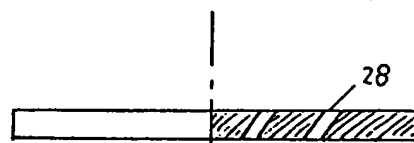


FIG 4

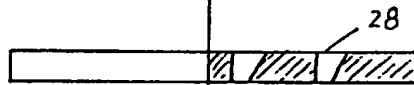


FIG 5

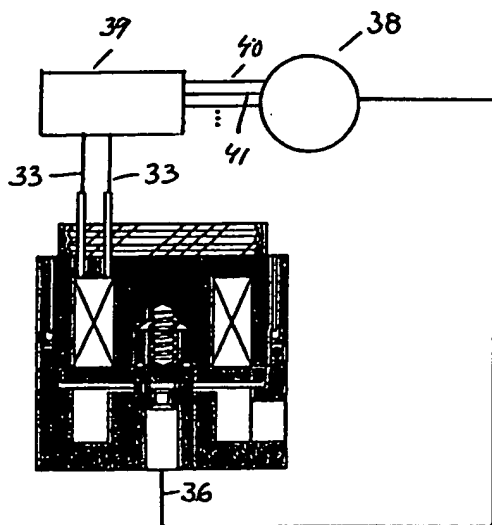


FIG 6